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Smyth, M., Pugsley, A., Hanna, G., Zacharopoulos, A., Mondol, J., & Besheer, A. (2017). *Experimental performance comparison of a Hybrid Photovoltaic/Solar Thermal (HyPV/T) Façade Module with a flat ICSSWH module*. Paper 56. Paper presented at First International Conference on Building Integrated Renewable Energy Systems (BIRES 2017), Dublin, Ireland.

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### Publication Status:

Published (in print/issue): 01/03/2017

### Document Version

Publisher's PDF, also known as Version of record

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# **Experimental performance comparison of a Hybrid Photovoltaic/Solar Thermal (HyPV/T) Façade Module with a flat ICSSWH module**

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## **Introduction**

The Energy Performance of Buildings Directive (EPBD) and Renewable Energy Framework Directive (REFD) require that Renewable Energy Systems (RES) are actively promoted in offsetting conventional fossil fuel use in buildings. A better appreciation of solar systems integration will directly support this objective, leading to an increased uptake in the application of renewables in buildings. By integrating these systems into the building elements (walls, roofs, etc.) not only means replacing a conventional building material (and associated costs), but also aesthetically integrating it into the building design leads to improved architectural integration.

Integrated Collector Storage Solar Water Heaters (ICSSWH) are simple, low cost solar devices and as such offer a suitable technology to partially meet the demands of the EPBD. Domestic solar thermal water heaters with integrated collectors and storage (ICS) offer cost and space efficiency benefits compared to conventional systems where the collectors and storage tanks are separate. These benefits would be further enhanced by combining the ICS with PV and locating all of the equipment within (rather than on the surface of) building façade and roof constructions. The development of these systems is detailed in Smyth et al (2006) along with their tendency to suffer significant ambient heat loss, especially at night-time and during non-collection periods. Many studies have been carried out focusing on the improvement of the thermal performance of ICSSWH systems, this paper presents the performance of a novel Hybrid Photovoltaic/Solar Thermal (HyPV/T) Façade Module compared with a flat ICSSWH module. The HyPV/T offers an interesting solution to mitigating the ICS heat loss problem, whilst providing a multi-functional device that addresses many of the wider issues facing solar devices deployed on buildings.

The ability for a single product to offer multiple functionality in a unique modular design that incorporates ICS technology, presents a huge commercial opportunity. The HyPV/T whilst offering a more cost effective solar investment will combine performance and quality and be fit for purpose, robust, visually appealing and exceptionally easy to install. These characteristics are expected in all premium solar collector-related products.

## **Description of the HyPV/T and flat ICSSWH modules**

A modular Hybrid Photovoltaic/Solar Thermal (HyPV/T) Façade technology that utilizes Integrated Collector Storage (ICS) solar technology, providing cost effective solar PV and thermal energy collection for direct use in the building, whilst providing significant thermal insulation has been developed and evaluated experimentally at Ulster University. The HyPV/T system, based upon a patented ICS solar thermal diode concept and shaped into a flat modular profile incorporating PV cells/module can provide space heating, domestic water heating and power generation. The complete system is designed to be compatible with traditional façade structures and fenestration framing arrangements, facilitating direct integration into new and retrofit building applications.

Two prototype units were designed, fabricated and investigated at Ulster. The performance of the rudimentary ICSSWH prototype without PV and without a transparent cover was initially quantified to establish a benchmark. Variants of the ICSSWH prototype were subsequently tested to determine the effect

of different transparent cover arrangements. The behaviour of the HyPV/T prototype was then characterised and compared against the results of the ICSSWH benchmark. Figure 2 depicts both completed HyPV/T (left) and flat ICSSWH (right) modules under the solar simulator ready for testing.

The flat ICSSWH unit(s) was based on an elliptical (Egyptian eye) profile with external dimensions 1000mm x 1000mm and 150mm deep. The ICS vessel was made from stainless steel sheet to form the collector/storage element which was supported by an internal exo-skeleton of 4 structural ribs giving a total storage volume of 35.65 litres. The absorbing surface of the ICS vessel was coated using matt black paint and the entire vessel enclosed in an outer wooden/polystyrene foam insulation case. The initial test was undertaken with an uncovered bare absorber surface. Subsequent tests were undertaken with a single cover (2mm Perspex) and a double cover (10mm transparent polypropylene twin-wall) over the absorber aperture.

The HyPV/T module was based around two principle elements; the thermal diode and water storage tank. Both elements were formed from folded and welded stainless steel sheet supported internally by an array of stainless steel tubular struts. A welded vacuum flange was added to enable working fluid addition and chamber evacuation through the top of the unit. A stainless steel manifold plate was welded to the bottom to enable connection of the evaporator wetting mechanism. The unit was approximately 1400mm long, 700mm wide and 190mm deep and had a storage volume of 100 litres. The photovoltaic-thermal absorber was formed on the diode's absorber-evaporator plate using matt black paint, double-sided adhesive foam tape, quartered mono-crystalline silicon cells (30 Bosch M-2BB high efficiency pseudo-square cells), transparent silicone resin and transparent Perspex cover plates. The series-parallel interconnected PV cells were connected to a  $8.6\Omega$  resistive load bank. Thermally insulating polystyrene foam was applied to the rear and sides of the prototype along with a transparent Perspex sheet to cover the absorber. Figure 1 gives a general arrangement of the HyPV/T module.

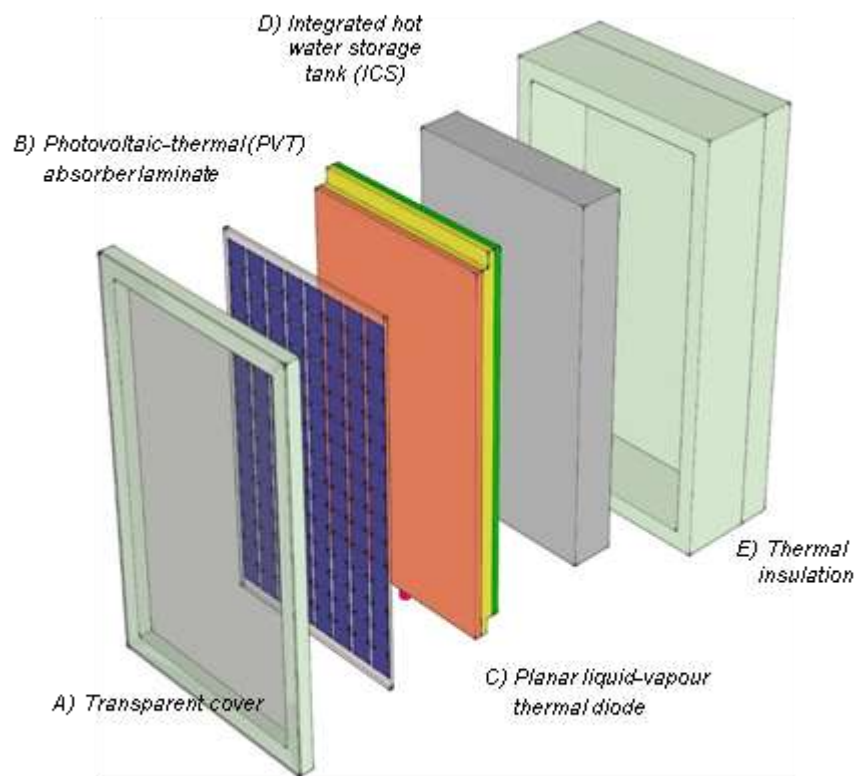


Figure 1: General arrangement of the Hybrid Photovoltaic/Solar Thermal (HyPV/T) module

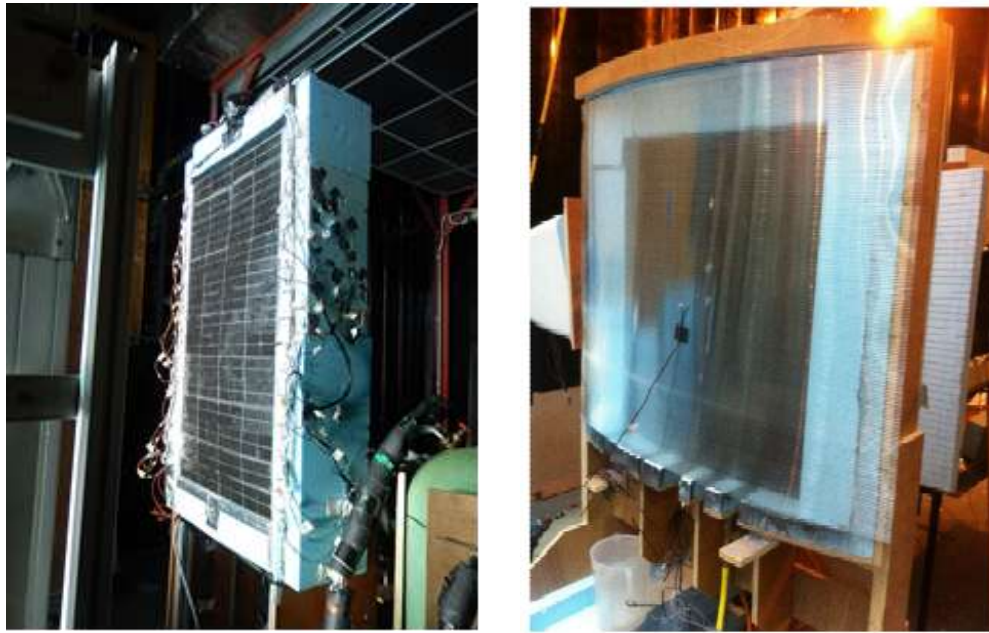


Figure 2: Images of the HyPV/T (left) and flat ICSSWH (right) modules under solar simulation testing

### **Description of the experimental facility**

The experimental performance of both prototypes was determined using the state-of-the-art solar simulation facility at Ulster University. The indoor solar simulator testing facility consisted of 35 high power metal halide lamps arranged in 7 rows of 5 lamps. Each lamp is equipped with a rotation symmetrical paraboloidal reflector to provide a light beam of high collimation. In order to achieve uniform distribution of light intensity on the test area, a lens is inserted in each lamp to widen the illumination of light. The combination of reflector-characteristics, lens and lamps ensures a realistic simulation of the beam path, spectrum and uniformity. The solar simulator control panel maintained the constant level light intensity automatically on the collector surface via a pyranometer mounted at the centre of the test plane. T-type copper-constantan thermocouples (error of  $\pm 0.5^{\circ}\text{C}$  between 0 and  $70^{\circ}\text{C}$ ) were used to measure the water storage temperatures within the various unit variants, surface temperatures and ambient air temperature. Water storage temperatures were measured at eight locations to record the variation of water temperature within the individual thermal stores. Temperatures were recorded through a Delta T logging device linked to a PC.

### **Description of the experimental procedure and performance analysis**

The collection and thermal retention performance characteristics for all units were determined under solar simulated conditions. System performance characterisation was based on BS ISO 9459-5:2007 (Anon, 2007a). No thermal storage draw off was performed during any of the tests. The tests were conducted over time periods of between 20 and 100 hours in order to determine the daily collection efficiency, typically consisting of 6 hours solar simulator exposure with an average solar intensity of  $600$  to  $800\text{W/m}^2$  followed by an 18 hour cool down period to determine thermal retention). At the beginning of each experiment each prototype's thermal store was re-filled and the test started with steady ambient temperatures and uniform tank temperatures.

The experimental performance of each unit was determined from the experimental data retrieved from the extensive testing programme. Thermal store, absorbing and external surfaces and external environment were analysed during testing to determine the system characteristics. Thermal store characterisation used the average normalised temperatures and stratification within the store both on collection and cool down to allow analysis of collection and thermal retention efficiencies and develop hypotheses for heat transfer mechanisms into and within the thermal store.

The total amount of simulated solar energy incident on the aperture area over the test period was determined using equation 1.

$$Q_{\text{incident}} = I_{\text{ave}} A_{\text{ap}} \Delta t \quad \{1\}$$

where

$$I_{\text{ave}} = \left( \int_{t_{\text{end}}}^{t_{\text{start}}} I(t) dt \right) / \Delta t \quad \{2\}$$

The recorded temperatures within the thermal stores were used to calculate mean temperatures in the entire storage vessel and using equation 3 the amount of thermal energy collected and retained was determined.

$$Q_{\text{col}} = mc_{\text{water}} (T_{\text{end}} - T_{\text{start}}) \quad \{3\}$$

The heat balance of the system with no draw-off is given by

$$mc_{\text{water}} (T_{\text{end}} - T_{\text{start}}) = I_{\text{ave}} (\eta_{\text{optical}}) A_{\text{ap}} \Delta t - UA_{\text{system}} \left[ \frac{(T_{\text{end}} - T_{\text{start}})}{2} - T_{\text{amb}} \right] \quad \{4\}$$

$$\eta_{\text{col}} = \frac{mc_{\text{water}} (T_{\text{end}} - T_{\text{start}})}{I_{\text{ave}} A_{\text{ap}} \Delta t} \quad \{5\}$$

$$\eta_{\text{col}} = (\eta_{\text{optical}}) - \frac{UA_{\text{system}} \left[ \left( \frac{T_{\text{end}} - T_{\text{start}}}{2} \right) - T_{\text{amb}} \right]}{I_{\text{ave}} A_{\text{ap}} \Delta t} \quad \{6\}$$

Equation 6 is superficially similar to the Hottel-Whillier-Bliss equation of solar collectors [Hottel and Whillier, 1958 Bliss, 1959], however this equation applies to a daily ICS system performance, as first proposed by Tripanagnostopoulos & Yianoulis (1992). Plots of efficiency versus  $\Delta T / I_{\text{ave}}$  can then be used to compare efficiencies where

$$\Delta T = \left( \frac{(T_{\text{end}} + T_{\text{start}})}{2} \right) - T_{\text{amb}} \quad \{7\}$$

Thermal retention efficiency ( $\eta_{\text{ret}}$ ) is determined by

$$\eta_{\text{ret}} = \left( \frac{m \times c (T_{\text{final}} - T_{\text{amb}})}{m \times c (T_{\text{initial,c}} - T_{\text{amb}})} \right) \times 100 \quad \{8\}$$

where  $T_{\text{initial,c}}$  is the average temperature at the start of the cool-down period,  $T_{\text{final}}$  is the average temperature at the end of the cooling period and  $T_{\text{amb}}$  is the average ambient temperature throughout the cool-down period.

### Experimental results and discussion

The experimental performance of the prototype Hybrid Photovoltaic/Solar Thermal (HyPV/T) unit has been determined and compared with the flat ICSSWH under constant indoor solar simulated conditions. The thermal performances of various modified HyPV/T and flat ICSSWH designs have been investigated and the thermal performance and collection efficiencies are presented.



The normalised average storage water temperatures for the various flat ICSSWH modules under solar simulation testing shown in figure 3 illustrate the heat retention benefits of an aperture cover for any BIST module. All the units exhibit an initial rapid rate of thermal collection, with the additional optical losses associated with the transparent covers having a limited but observable impact on the final store temperature. This translated into instantaneous thermal collection efficiencies under zero temperature rise conditions (equalised to ambient) of 65%, 64% and 62%, for the unglazed (bare), single and double glazed units, respectively. During cool-down, the bare unit lost all of the collected thermal energy by the end of the test whilst the covered variants exhibited a positive gain of approximately 5°C in the thermal store. The inclusion of the second transparent cover proved to have small benefit (approximately 2°C) in improving thermal retention. The ambient temperatures were similar during all tests. The average storage water temperatures observed over the cool-down period are also mirrored in the respective heat retention efficiencies. The unglazed (bare) unit had a retention efficiency of 8.3% whilst the unit with a single transparent cover was 23.6% and double glazed unit was 28%.

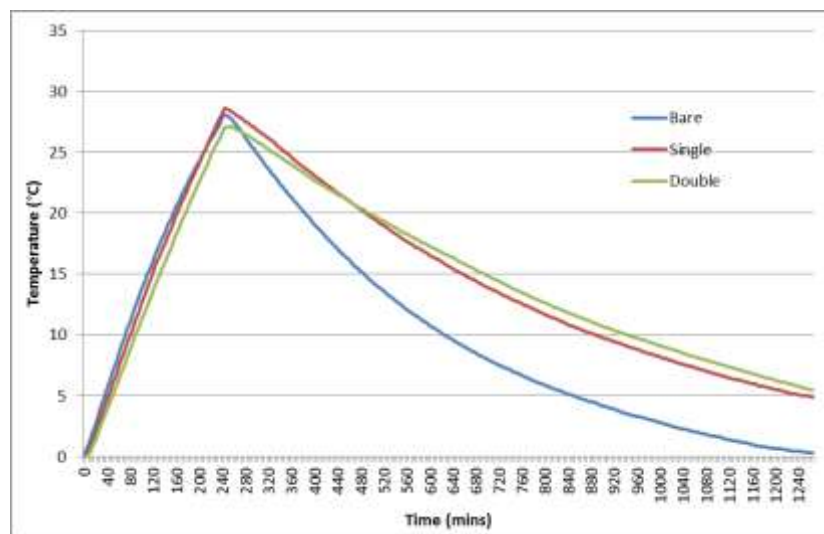


Figure 3: Normalised (to starting temperature) average storage water temperatures for the various flat ICSSWH modules under solar simulation testing

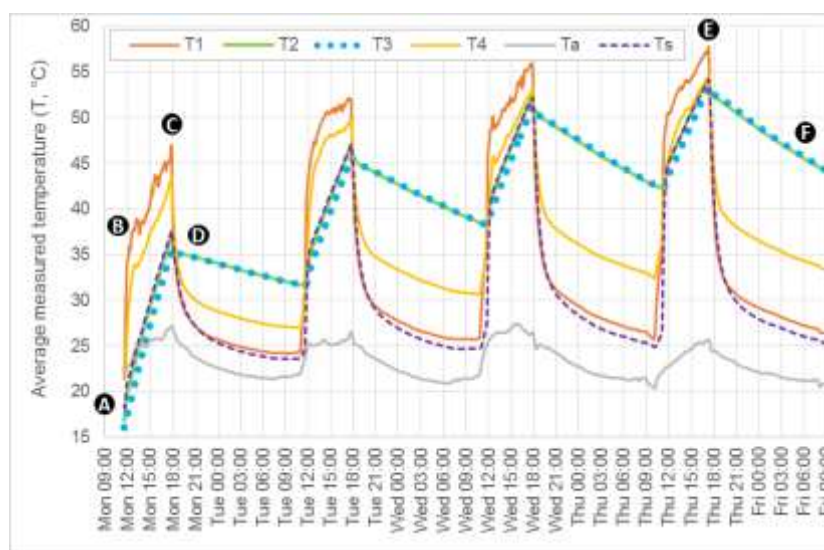


Figure 4: Average measured temperatures for the unglazed HyPV/T unit under solar simulation testing

Figure 4 shows the temperature time history during the 4 day collection/cool-down testing under  $I_{ave} = 610\text{W/m}^2$  irradiance for the HyPV/T unit with no cover. At the start of the test the water storage tank ( $T_3$ ) was at  $16^\circ\text{C}$  and was similar to the other unit component temperatures ( $T_1$ , absorber plate;  $T_2$ , condenser;  $T_4$ , thermal diode sidewalls;  $T_s$ , working fluid sump). The absorber, working fluid sump, and thermal diode sidewall temperatures increased rapidly as soon as the absorber was exposed to the solar simulator (at Point “A”) and began to rise rapidly (at Point “B”) after about half an hour and at a lower rate thereafter. All parts of the prototype steadily increased in temperature at a broadly similar rate during the majority of the 6 hour collection period. When the solar simulator was switched off (at Point “C”) the condenser ( $T_2$ ) and tank ( $T_3$ ) temperatures dropped steadily whereas the absorber, sump, and sidewalls cooled rapidly for  $\sim 2$  hours (until Point “D”). For the latter part of the 18 hour cool-down period all parts of the prototype cooled at a similar rate. Each of the 4 collection periods and 4 cool-down periods followed a similar pattern (albeit with differing temperatures) with ‘dawn and dusk’ temperatures increasing on subsequent days until the tank temperature reached a maximum of  $T_3=53^\circ\text{C}$  before cooling overnight to  $T_3=44^\circ\text{C}$ . Forward mode temperature difference across the thermal diode was typically  $\Delta T_{1-2}\sim 15^\circ\text{C}$  at the start of the test (at Point “A”) and steadily fell to  $\Delta T_{1-2}\sim 3^\circ\text{C}$  (at Point “E”) indicating that diode thermal conductance increased with temperature, as expected. The reverse mode temperature difference across the thermal diode was typically  $\Delta T_{1-2}\sim 8^\circ\text{C}$  at the start of the test (at Point “A”) and  $\Delta T_{1-2}\sim 18^\circ\text{C}$  at the end (at Point “F”).

The unglazed HyPV/T achieved thermal and PV efficiencies under zero temperature rise conditions (equalised to ambient) of 56% and 9%, respectively. During the continuous 4 day collection/heat retention test period (with no water draw offs), the daily collection efficiency varied from 56% on the first day (starting with a cold tank) to 38% on the last (starting with warm tank). Overnight (18hr) heat retention efficiencies were  $65\pm 5\%$  and the overall heat loss coefficient was  $1.1\text{ W/m}^2\text{K}$ .

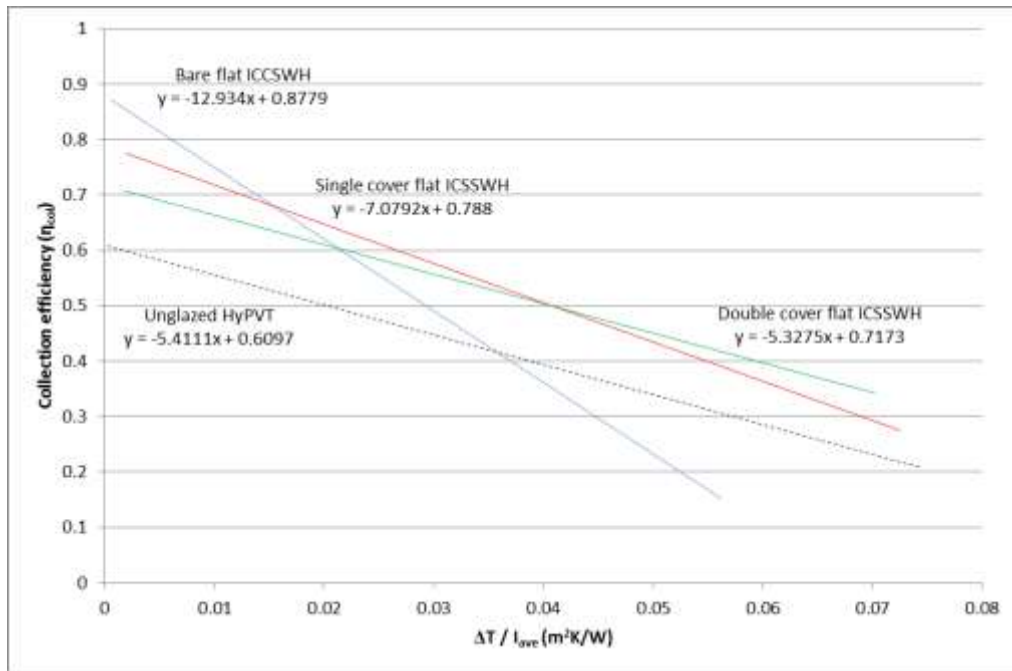


Figure 5: Plots of efficiency versus  $\Delta T / I_{ave}$  for various HyPV/T and flat ICSSWH (right) modules under solar simulation testing

Figure 5 shows system efficiency plots versus  $\Delta T / I_{ave}$  (using the Hottel-Whiller-Bliss form based on equation 6) for the unglazed HyPV/T against the 3 flat ICSSWH variants (unglazed, single and double glazed apertures). The unglazed flat ICSSWH exhibits the highest optical efficiency but has a steeper ‘efficiency curve’ gradient due to a greater heat loss. Both the glazed flat ICSSWH units have similar ‘efficiency curves’ with the single glazed unit exhibiting a slightly better optical performance but with more heat loss. The unglazed HyPV/T, due to the PV element has a much lower apparent optical efficiency but due to the thermal diode and the low emissivity characteristics of the PV, has better heat loss characteristics.

## Conclusion

A modular Hybrid Photovoltaic/Solar Thermal (HyPV/T) Façade technology that utilizes Integrated Collector Storage (ICS) solar technology, potentially providing cost effective solar PV and thermal energy collection for direct use in the building, whilst providing significant thermal insulation has been developed and evaluated experimentally under the solar simulator facility at Ulster University. The HyPV/T system, based upon a patented ICS solar thermal diode concept has been tested and its thermal performance analysed and compared with a series of 'traditional' flat ICSSWH units.

The daily thermal collection efficiencies (equalised to ambient) for both systems were determined. As expected, the 'traditional' flat ICSSWH units (unglazed, single and double glazed units) performed better than the unglazed HyPV/T, by 5 to 10%. This is also evident from the system efficiency plots shown in figure 5. However, when the additional electrical power produced by the HyPV/T is included, the overall system collection efficiencies are more equal. Furthermore, a glazed version of the HyPV/T could potentially improve upon the measured thermal efficiencies (albeit with a corresponding drop in electrical conversion). The heat retention performance however is much more promising. The unglazed (bare) ICS unit had a retention efficiency of 8.3% whilst the ICS unit with a single transparent cover was 23.6% and double glazed unit was 28%. The HyPV/T heat retention efficiencies were approximately 65% over the same cool-down period.

The work presented in this paper illustrates that it is possible to design and develop a modern ICSSWH unit that can collect significant quantities of solar thermal energy and retain the energy over extended periods of non-collection. Whilst the thermal performance of the proposed HyPV/T thermal diode concept compares favourably with other forms of ICSSWH units, the multi-functional design offers additional benefits. Shaped into a flat modular profile incorporating PV cells/module the unit can provide improved thermal insulation linked to space heating with domestic water heating and power generation supplied directly to a building. The complete system is designed to be compatible with traditional façade structures and fenestration framing arrangements, facilitating direct integration into new and retrofit building applications.

## Nomenclature

$A_{ap}$	aperture area ( $m^2$ )
$c$	specific heat capacity of water (J/kgK)
$I_{ave}$	insolation ( $W/m^2$ )
$m$	mass of water (kg)
$Q_{incident}$	incident solar thermal energy (J)
$T$	temperature ( $^{\circ}C/K$ )
$UA_{system}$	heat loss coefficient of system (W/K)
$\Delta t$	time (secs)
$\Delta T$	temperature difference ( $^{\circ}C/K$ )
$\eta_{col}$	collection efficiency
$\eta_{optical}$	optical efficiency
$\eta_{ret}$	retention efficiency

## Subscripts

<b>a/amb</b>	average ambient temperature
<b>end</b>	average temperature at end of heating period
<b>final</b>	average final water temperature at end of cooling period
<b>initial,c</b>	average initial water temperature at start of cooling period
<b>start</b>	average temperature at start of heating period



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